# SECOND SEMESTER DIPLOMA EXAMINATION IN ENGINEERING <br> AND TECHNOLOGY <br> (Common to AR / AU / CE / ME / MT / TD / WP) 

## ENGINEERING MECHANICS MODEL QUESTION PAPER

Time: 3 hours
Maximum Marks: 75

PART A
I. Answer all questions in one word or one sentence. Each question carries one mark.
( $9 \times 1=9$ Marks)

| 1 | An infinite straight line along which the force acts is called .................... | MO1.01 | R |
| :---: | :---: | :---: | :---: |
| 2 | A force of 10 kN is acting at $60 \square$ with vertical. Determine the horizontal and vertical component of force. | MO1.03 | U |
| 3 | Define cantilever beam | MO2.01 | R |
| 4 | No. of restraints in fixed beam is............ | MO2.01 | U |
| 5 | The position centroid of a triangular lamina from the base is ....... | MO3.01 | R |
| 6 | Name the moment of inertia about an axis(Izz) which is perpendicular to other the mutually perpendicular axes Ixx and Iyy. | MO3.05 | U |
| 7 | Internal resistance offered by a body against external loading is called. $\qquad$ | MO4.01 | R |
| 8 | Ratio of lateral strain to linear strain is........... | MO4.05 | R |
| 9 | The maximum value of static friction comes into play when a body just starts to slide over another is $\qquad$ | MO4.06 | R |

## PART B

II. Answer any eight questions from the following. Each question carries $\mathbf{3}$ marks

$$
\text { ( } 8 \times 3=24 \text { Marks) }
$$

| 1 | Differentiate scalar quantity and vector quantity | MO 1.02 | R |
| :---: | :--- | :---: | :---: |


| 2 | Determine the magnitude of the reaction force R | MO1.03 | U |
| :---: | :---: | :---: | :---: |
| 3 | State Varignons theorem | MO1.03 | R |
| 4 | Define simply supported and cantilever beam. | MO2.01 | R |
| 5 | Sate the laws of friction | MO2.05 | R |
| 6 | Illustrate the center of gravity of the following solid bodies <br> a) Hemisphere <br> b) Cone | MO3.01 | R |
| 7 | State perpendicular axis theorem | MO3.04 | R |
| 8 | Draw the stress strain curve of steel and explain the terms <br> a) Limit of proportionality <br> b) Ultimate stress | MO4.02 | R |
| 9 | Explain the following properties <br> a) Elasticity <br> b) Plasticity <br> c) Toughness | MO4.04 | R |
| 10 | The value of modulus of elasticity and poisons ratio of an alloy body is 150 GPA and 0.25 respectively. Determine the value of bulk modulus of the alloy. | MO4.05 | U |

## PART C

## Answer all questions. Each question carries seven marks

$$
\text { ( } 6 \times 7=42 \text { Marks) }
$$

| III | A boat is moved uniformly along a canal by two horses pulling <br> with forces $\mathrm{P}=890 \mathrm{~N}$ and $\mathrm{Q}=1068 \mathrm{~N}$ acting under an angle $\alpha=$ <br> $60^{\circ}$. Determine the magnitude of the resultant pull on the boat and <br> the angles $\beta$ and $v$. | MO1.03 | A |
| :--- | :--- | :--- | :--- |



\begin{tabular}{|c|c|c|c|}
\hline \& \(25^{\circ}\) to the horizontal, find the weight of the body and coefficient of friction. \& \& \\
\hline IX \& \begin{tabular}{l}
OR \\
Determine the moment of inertia about the centroidal axes of given I section beam given below. All dimensions in mm
\end{tabular} \& MO3.02

MO3.04 \& | U |
| :---: |
|  |
|  |
|  |
|  | <br>

\hline XI \& | Determine the centroid of given section |
| :--- |
| OR | \& MO3.02 \& U <br>

\hline XII \& Find out the moment of inertia of the shaded area in the figure about the base. \& MO3.04 \& A <br>
\hline
\end{tabular}


## ANSWER KEY

## ENGINEERING MECHANICS MODEL QUESTION PAPER

PART A

Answer all the following questions
( $9 \times 1=9$ Marks)
I

| Q.No | Answer | Split up | Total Mark |
| :---: | :--- | :--- | :---: |
| 1 | Line of action of force | 1 | 1 |
| 2 | Vertical component = 10 Sin60 <br> Horizontal component = 10 Cos60 | 1 | 1 |
| 3 | A beam with one end fixed and other end free | 1 | 1 |
| 4 | 3 | 1 | 1 |
| 5 | h/3 | 1 | 1 |
| 6 | Polar moment of inertia | 1 | 1 |
| 7 | Stress | 1 | 1 |
| 8 | Poisons ratio | 1 | 1 |
| 9 | Limiting friction | 1 | 1 |

II.

PART B

| Q.No | Answer | Split up | Total Mark |
| :---: | :--- | :--- | :---: |
| 1 | Scalars: only magnitude is associated. | 1 |  |
|  | Ex: time, volume, density, speed, energy, mass |  |  |
| Vectors: possess direction as well as magnitude, and must <br> obey the parallelogram law of addition (and the triangle <br> law). <br> Ex: displacement, velocity, acceleration, force, moment, <br> momentum | 1 | 1 | 3 |


| 2 | $\frac{T}{\sin 31.4}=\frac{R}{\sin 110}=\frac{98.1 \mathrm{~N}}{\sin 38.6^{\circ}}$ $\begin{aligned} & T=81.9 \mathrm{~N} \\ & R=147.8 \mathrm{~N} \end{aligned}$ | $2$ <br> 1 | 3 |
| :---: | :---: | :---: | :---: |
| 3 | The moment of the resultant of two concurrent forces with respect to a center in their plane is equal to the algebraic sum of the moments of the components with respect to some centre. | 3 | 3 |
| 4 | simply supported beam is supported at both ends. One end of the beam is supported by hinge support and the other one by roller support. <br> Cantilever beam is a structural member of which one end is fixed and other end is free | $1.5$ $1.5$ | 3 |
| 5 | 1. Friction always acts in the direction opposite to the motion or impending motion. <br> 2. The limiting friction is directly proportional to the normal reaction. <br> 3. Until the motion starts the static frictional force adjust itself to just balance the force tending to produce motion. <br> 4. Friction is independent of the area of contact between the two surfaces but depends on the roughness of the surface. <br> 5. Kinetic friction also bears a constant ratio with normal reaction but this ratio is slightly less than that in the case of limiting friction. <br> 6. For moderate speeds, friction remains constant but it decreases slightly for higher speeds. |  | 3 |
| 6 | a) | 1.5 |  |


|  |  | 1.5 | 3 |
| :---: | :---: | :---: | :---: |
| 7 | The theorem states that the moment of inertia of a plane lamina about any two mutually perpendicular axes in its plane and intersecting each other at the point where the perpendicular axis passes through it. $\mathrm{I}_{\mathrm{zz}}=\mathrm{I}_{\mathrm{yy}}+\mathrm{I}_{\mathrm{xx}}$ | 3 | 3 |
| 8 |  <br> a)Limit of proportionality <br> Point A, in this limit the stress is directly proportional to strain $\{\sigma \propto \mathrm{e}\}$, that means the steel rod obeys 'Hooke's law <br> b)Ultimate stress | 1 |  |


|  | Point is 'E' which is called as ultimate stress or ultimate <br> strength point. Ultimate stress is the maximum stress the rod <br> can with stand, thus this portion is called a strain hardening. |  |  |
| :---: | :---: | :--- | :--- |
| 9 | a)Elasticity: The ability of an object or material to <br> resume or regain its normal shape or original shape <br> after being stretched or compressed called Elasticity. <br> b)Plasticity: he quality of being easily shaped or <br> molded called Plasticity. <br> c)Toughness: t is the state of being strong enough in <br> order to withstand adverse conditions or rough <br> handling called Toughness | 1 | 3 |
| 10 | $\mathrm{~K}=\mathrm{E} / 3(1-2 / \mathrm{m})=\left(150 \times 10^{3}\right) / 3(1-2 \times 0.25)=100 \mathrm{GPa}$ | $1+1+1$ | 3 |

## PART C

III \begin{tabular}{lll}

| $\mathrm{P}=890 \mathrm{~N}, \alpha=60^{\circ}$ |
| :--- |
| $\mathrm{Q}=1068 \mathrm{~N}$ |
| $R=\sqrt{\left(P^{2}+Q^{2}+2 P Q \cos \alpha\right)}$ |
| $=\sqrt{\left(890^{2}+1068^{2}+2 \times 890 \times 1068 \times 0.5\right)}$ |
| $=1698.01 \mathrm{~N}$ | \& \& 2

\end{tabular}

\begin{tabular}{|c|c|c|}
\hline \& \[
\begin{aligned}
\& \frac{Q}{\sin \beta}=\frac{P}{\sin v}=\frac{R}{\sin (\pi-\alpha)} \\
\& \sin \beta=\frac{Q \sin \alpha}{R} \\
\& =\frac{1068 \times \sin 60^{\circ}}{1698.01} \\
\& =33^{\square} \\
\& \sin v=\frac{P \sin \alpha}{R} \\
\& =\frac{890 \times \sin 60^{\square}}{1698.01} \\
\& =27^{\square}
\end{aligned}
\] \& 2 \\
\hline \& OR \& \\
\hline IV \& \begin{tabular}{l}
In triangle \(\mathrm{ABC} \operatorname{Sin} \varphi=5 / 10=0.5\)
\[
\Phi=30^{\circ}
\] \\
Consider roller in equilibrium and apply condition of equilibrium
\[
\Sigma \mathrm{H}=0 \text { and } \Sigma \mathrm{V}=0
\] \\
Resolving forces horizontally
\[
\mathrm{FCos} 30-200=0
\] \\
Force in bar AB , \(\mathrm{F}=200 / \operatorname{Cos} 30=230.9 \mathrm{~N}\) \\
Resolving forces vertically \\
Rc -FSin30-100=0
\[
R c=230.9 \operatorname{Sin} 30+100=215.47 \mathrm{~N}
\]
\end{tabular} \& 1

2
1
1 <br>
\hline
\end{tabular}

|  |  |  | 7 |
| :--- | :--- | :--- | :--- |
| V |  |  |  |




\begin{tabular}{|c|c|c|c|}
\hline \& \begin{tabular}{l}
Resolving vertically,
\[
\Sigma \mathrm{V}=0
\] \\
\(\mathrm{S}_{5} \sin 45=\mathrm{S}_{2} \sin 45, \mathrm{~S}_{5}=113.137 \mathrm{kN}\) (Compression) \\
Resolving Horizontally,
\[
\begin{aligned}
\Sigma \mathrm{H} \& =0 \\
\mathrm{~S}_{6} \& =\mathrm{S}_{5} \operatorname{Cos} 45+\mathrm{S}_{2} \operatorname{Cos} 45 \\
\mathrm{~S}_{6} \& =113.137 \operatorname{Cos} 45+56.56 \operatorname{Cos} 45 \\
\& =120 \mathrm{KN}
\end{aligned}
\]
\end{tabular} \& 1

1 \& 1

1 <br>

\hline VIII \& |  |
| :--- |
| Pull required, $\mathrm{P}=20 \mathrm{~N}$ |
| Inclination of pull, $\Phi=25^{\circ}$ |
| Push required $\mathrm{P}_{1}=25 \mathrm{~N}$ |
| Inclination of push, $\Phi=25^{\circ}$ |
| Resolving forces along the plane $\mathrm{F}=20 \cos 25^{\circ}$ |
| Resolving forces normal to plane $\mathrm{R}+20 \operatorname{Sin} 25=\mathrm{W}$ | \& 1 \& <br>

\hline
\end{tabular}






$$
I_{A B}=I_{G}+A \bar{h}^{\angle}
$$




| XIII | $\begin{aligned} & \mathrm{L}=150 \mathrm{~cm} \\ & \mathrm{D}=2 \mathrm{~cm} \end{aligned}$ <br> Area $=\pi / 4(20)^{2}=100 \pi \mathrm{~cm}^{2}$ $\begin{aligned} & \mathrm{P}=20 \mathrm{kN} \\ & \mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2} \end{aligned}$ <br> (i) Stress $=\mathrm{P} / \mathrm{A}=20000 / 100 \pi=63.662 \mathrm{~N} / \mathrm{mm}^{2}$ <br> (ii) Strain $e=\sigma / E=63.662 / 2 \times 10^{5}=0.000318$ <br> (iii) Elongation $\mathrm{dL}=\mathrm{e} \times \mathrm{L}=0.000318 \times 150=0.0477 \mathrm{~cm}$ | 2 2 2 |  |
| :---: | :---: | :---: | :---: |
| XIV | $\begin{aligned} & \mathrm{L}=4 \mathrm{~m} \\ & \mathrm{~b}=30 \mathrm{~mm} \\ & \mathrm{t}=20 \mathrm{~mm} \\ & \mathrm{~A}=\mathrm{b} \times \mathrm{t} \\ & =30 \times 20=600 \mathrm{~mm}^{2} \\ & \mathrm{P}=30 \mathrm{kN}=30000 \mathrm{~N} \\ & \mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2} \\ & \mu=0.3 \end{aligned}$ <br> Longitudinal strain $\mathrm{e}=\mathrm{P} / \mathrm{AE}$ $\begin{aligned} & \quad=30000 / 600 \times 2 \times 10^{5}=0.00025 \\ & e=d L / L=0.00025 \\ & d L=0.00025 \times 4000=1.0 \mathrm{~mm} \end{aligned}$ <br> Poisson's ratio, $\mu=$ lateral strain / longitudinal strain $\text { lateral strain }=0.3 \times 0.00025=0.000075$ $\mathrm{db}=\mathrm{b} \times \text { lateral strain }=30 \times 0.0000075=0.00225$ <br> mm. $\mathrm{dt}=\mathrm{t} \times \text { lateral strain }=20 \times 0.0000075=0.0015 \mathrm{~mm} .$ | 1 1 1 1 1 1 1 1 1 1 | 7 |

